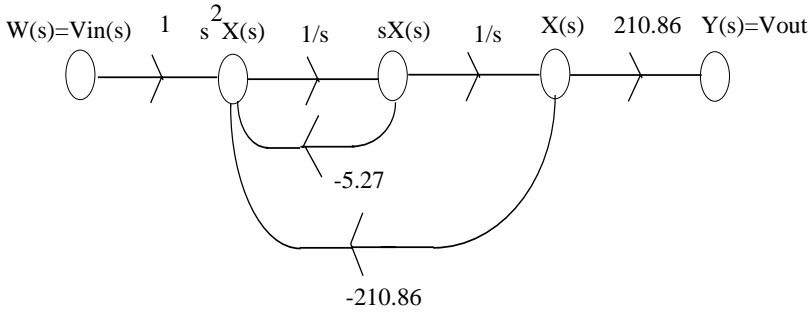


1a) From Lab 1, our transfer function is $\frac{Y(s)}{W(s)} = \frac{V_{out}(s)}{V_{in}(s)} = \frac{210.86}{s^2 + 5.2711s + 210.86}$. Using phase variables, we obtain the following SFG:



$$\Rightarrow \dot{x} = Ax + Bw = \begin{bmatrix} 0 & 1 \\ -210.86 & -5.27 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} w$$

$$y = V_{out} = Cx + Dw = [210.86 \quad 0]x$$

1b) Note that $y=V_{out}=210.86x_1$ and $V_g = K_g \dot{\theta}_m = K_g N_a \dot{\theta}_{out} = K_g (9) \dot{\theta}_{out} = K_g (9) \dot{V}_{out} / K_p = \frac{K_g (9)}{K_p} 210.86 \dot{x}_1$

But from our state variable model, $\dot{x}_1 = x_2$ and from Lab 1, $K_g = 0.1842$ vs/r and $K_p=5.1394$ v/r. Hence, $V_g=(0.1842)(9)(210.86)(x_2)/(5.1394) = 68.0164x_2$. Thus, we can write the following similarity transformation which relates our state variables in part a) to the (measurable) physical state variables, $[V_{out} \ V_g]^T$:

$$\begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = Tz = T \begin{bmatrix} V_{out} \\ V_g \end{bmatrix} = \begin{bmatrix} 1/210.86 & 0 \\ 0 & 1/68.0164 \end{bmatrix} \begin{bmatrix} V_{out} \\ V_g \end{bmatrix}$$

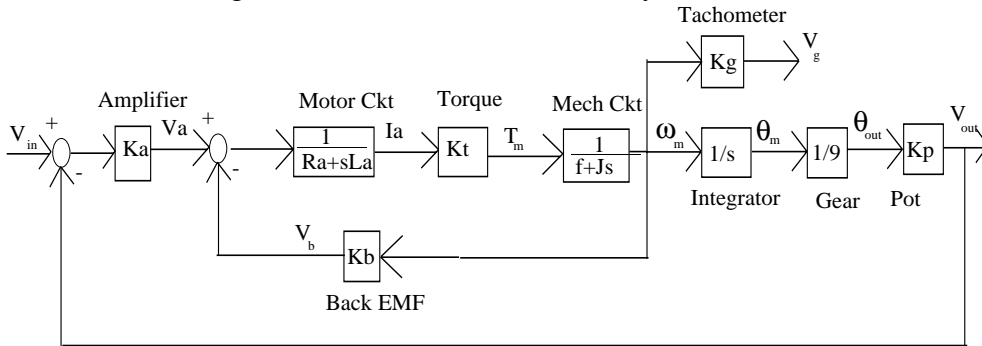
Plugging this similarity transformation into our state

model from part a) produces the following new state variable model in our physical state variables, $[V_{out} \ V_g]^T$:

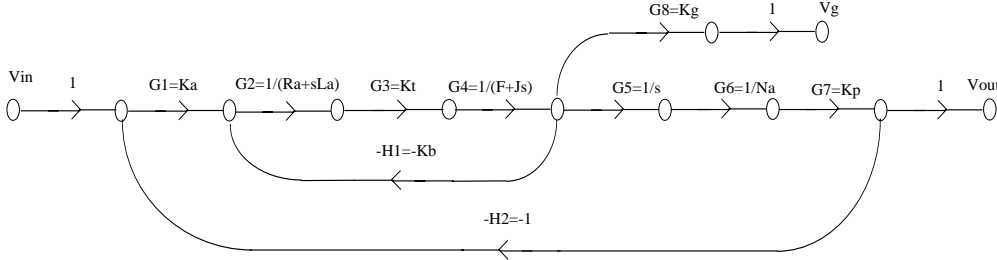
$$\dot{z} = T^{-1}ATz + T^{-1}Bw \Rightarrow \begin{bmatrix} \dot{V}_{out} \\ \dot{V}_g \end{bmatrix} = \begin{bmatrix} 0 & 3.1 \\ -68.014 & -5.27 \end{bmatrix} \begin{bmatrix} V_{out} \\ V_g \end{bmatrix} + \begin{bmatrix} 0 \\ 68.0164 \end{bmatrix} V_{in}$$

$$y = CTz \Rightarrow y = [1 \quad 0] \begin{bmatrix} V_{out} \\ V_g \end{bmatrix}$$

2a) The block diagram for the Motomatic Servo-system is:



2b) To find the overall transfer function, we can first convert this to the following SFG:



By Mason's SFG Formula, the overall transfer function is:

$$\frac{V_{out}(s)}{V_{in}(s)} = \sum \frac{T_n \Delta_n}{\Delta} = \frac{T_1 \Delta_1}{\Delta} \text{ where } T_1 = G_1 G_2 G_3 G_4 G_5 G_6 G_7 \text{ and}$$

$$\Delta = 1 - \sum L_1' s + \sum L_2' s - \sum L_3' s + \dots$$

$$= 1 - [G_2 G_3 G_4 (-H_1) + G_1 G_2 G_3 G_4 G_5 G_6 G_7 (-H_2)] + 0$$

$$\text{Thus, } \Delta_1 = 1 \text{ and } \frac{V_{out}(s)}{V_{in}(s)} = \frac{G_1 G_2 G_3 G_4 G_5 G_6 G_7}{1 + G_2 G_3 G_4 H_1 + G_1 G_2 G_3 G_4 H G_5 G_6 G_7 H_2}$$

$$\text{or } \frac{V_{out}}{V_{in}} = \frac{K_a K_T K_P / 9}{L_a J s^3 + (L_a F + J R_a) s^2 + (R_a F + K_b K_t) s + K_a K_T K_P / 9}$$

2c) From the SFG and Mason's Gain Formula, we see that:

$$\frac{V_g(s)}{V_{in}(s)} = \sum \frac{T_n \Delta_n}{\Delta} = \frac{T_1 \Delta_1}{\Delta} \text{ where } T_1 = G_1 G_2 G_3 G_4 G_8 \text{ and}$$

$$\Delta = 1 - \sum L_1' s + \sum L_2' s - \sum L_3' s + \dots = 1 - [G_2 G_3 G_4 (-H_1)] + 0$$

$$\text{Thus, } \Delta_1 = 1 \text{ and } \frac{V_g(s)}{V_{in}(s)} = \frac{G_1 G_2 G_3 G_4 G_8}{1 + G_2 G_3 G_4 H_1}$$

$$\text{or } \frac{V_g}{V_{in}} = \frac{K_a K_T K_g}{J L_a s^2 + (L_a F + J_a R_a) s + K_b K_T + R_a F}$$

2d) From the block diagram, $\frac{I_a(s)}{V_a(s) - V_b(s)} = \frac{1}{R_a + L_a s}$. If we hold the motor shaft and prevent it from turning,

then we force V_b to be zero. Thus, $\frac{I_a(s)}{V_a(s)} = \frac{1}{R_a + L_a s}$. If V_a is sinusoidal, then we obtain two equations using

$$\text{the magnitude and phase of the response: } \left| \frac{I_a(j\omega)}{V_a(j\omega)} \right| = \frac{1}{\sqrt{R_a^2 + (L_a \omega)^2}} \text{ and } \angle \frac{I_a(j\omega)}{V_a(j\omega)} = \tan^{-1} \left(\frac{\omega L_a}{R_a} \right)$$